

**IN THE SPECIFICATION**

Please amend paragraph [0021] as follows:

Turning now to Figure 2, a newly designed gas shield 36 in accordance with an exemplary embodiment of this invention, includes an annular ring body 38 that includes a radial flange 40 provided with mounting gussets 42 adjacent a first free end of the ring body. These gussets 42 are used to secure the gas shield to the section plate 26 on the inlet side of the fan. The gas shield terminates in the radial outward direction at a first free edge 44 of a relatively larger diameter. The gas shield extends radially inwardly through a curved air inlet portion 45 and then axially where it defines a center opening 46 that surrounds the fan blades. The axial portion 48 defines a second smaller diameter and includes a seal insert recess 50 for a toothed or other seal element. The shield is then bent back toward the armature bars (see Figure 3) so as to create an aerodynamically smooth, curved outlet portion 52 on the exit or outlet side of the fan, terminating at a second free end 54 of a third diameter greater than the second diameter but smaller than the first diameter. The gas shield thus directs the cooling gas flow along the aerodynamically smooth outlet portion 52 of the shield to cool the armature bars. The gap formed between the gas shield tip and the armature bar edge (see Figure 3) can be used to control the flow rates through the main body of the armature bar and towards the series loop caps. The gas shield profile at the fan flow exit is primarily determined by two design parameters: radius  $R$  and angle  $\theta$ . The values of  $R$  and  $\theta$  may vary according

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to the generator type. The aerodynamically smooth surface at the gas shield tip reduces the flow drag course and windage loss.